

Principal Stress & Maximal Shear Stress

2-Dimensional

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\tau_{max} = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\sigma_{x'} = \frac{\sigma_x + \sigma_y}{2} + \left(\frac{\sigma_x - \sigma_y}{2}\right) \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\sigma_{y'} = \frac{\sigma_x + \sigma_y}{2} - \left(\frac{\sigma_x - \sigma_y}{2}\right) \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\tau_{x'y'} = -\left(\frac{\sigma_x - \sigma_y}{2}\right) \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$\theta = \frac{1}{2} \tan^{-1} \left(\frac{2\tau_{xy}}{\sigma_x - \sigma_y} \right)$$

$$I_{xx} = \int y^2 dA, I_{yy} = \int x^2 dA, I_{xy} = \int xy dA, I_{xx'} = I_{xx} \cos^2 2\theta - I_{xy} \sin 2\theta + I_{yy} \sin^2 2\theta$$

3-Dimensional

Determinants Equations (Stress Invariants)

$$I_1 = \sigma_x + \sigma_y + \sigma_z$$

$$I_2 = \sigma_x \sigma_y + \sigma_y \sigma_z + \sigma_z \sigma_x - \tau_{xy}^2 - \tau_{yz}^2 - \tau_{zx}^2$$

$$I_3 = \sigma_x \sigma_y \sigma_z + 2\tau_{xy} \tau_{yz} \tau_{zx} - \sigma_x \tau_{xy}^2 - \sigma_y \tau_{yz}^2 - \sigma_z \tau_{zx}^2 - 3\tau_{xy} \tau_{yz} \tau_{zx}$$

$$I_1^3 - 3I_1 I_2 + 3I_3 = 0 \Rightarrow \sigma_1, \sigma_2, \sigma_3$$

Principle stress of 1, 2, 3, where

$$\sigma_{max} = \frac{I_1 + \sqrt{I_1^2 - 3I_2}}{3}$$

See ex. for principal direction

$$\sigma_{min} = \frac{I_1 - \sqrt{I_1^2 - 3I_2}}{3}$$

$$\sigma_{int} = \frac{I_1 + \sqrt{I_1^2 - 3I_2}}{3}$$

Centroidal Bending

$$\sigma_{max} = \frac{M y_c}{I_x}$$

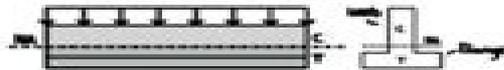
c is distance from NB to point of interest; y is taken for counter-clock

I_x is the MCI about axis

bending stress will be MCI (perpendicular axis)

M is axial (perpendicular) stress distribution

tensile bending \rightarrow positive, compression bending \rightarrow negative

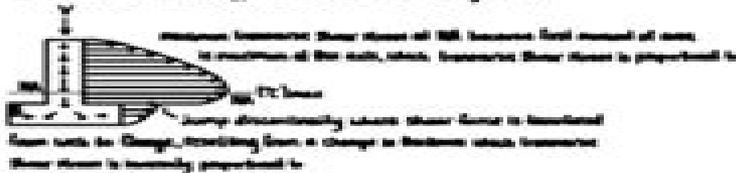


Transverse Shear

$$\tau = \frac{VQ}{It}$$

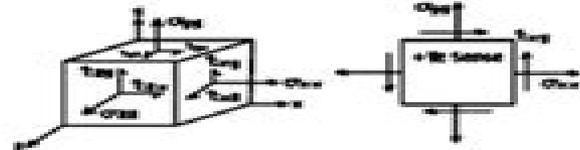
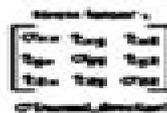
Trans \rightarrow trans, for direction along axis but I/Q are with respect to

I/Q will be MCI axis opposite to V (another axis (perpendicular))



Twisting

$$\tau = \frac{T\rho}{J}$$



Plane Sectional Properties

$$Centroid: \bar{y} = \frac{\sum y_i A_i}{\sum A_i}$$

Second Moment of Area

$$I_x = \sum y_i^2 A_i + A_i \bar{y}_i^2$$

$$I_y = \sum x_i^2 A_i + A_i \bar{x}_i^2$$

If I_{xy} , then (b) is face perspective of y -axis

If I_{xy} , $\theta = \frac{1}{2} \tan^{-1} \left(\frac{2I_{xy}}{I_x - I_y} \right)$

Area Moment of Inertia

$$J = \sum r_i^2 A_i$$

$I_x = \sum y_i^2 A_i$ if bar is used as a shaft; hollow use $I_o - I_h$

First Moment of Area

moment of area above or below point of interest

$$Q = A \bar{y}$$

where A is the remaining cross sectional area above or below that point

and \bar{y} is the distance between the centroid of that area to the centroid of the entire cross section

2-Dimensional Plane Stress & Strain

Stress

(1) — one dimension compared to other two dimensions is very small

(2) — like a plate or a shell, stress with large constant

(3) — total has to be in the plane of the plate, etc., same way it flows out.

Strain

(1) — one dimension compared to other two dimensions is very large

(2) — symmetric loading condition

Stress-Strain

$$\begin{bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} \\ C_{21} & C_{22} & C_{23} \\ C_{31} & C_{32} & C_{33} \end{bmatrix} \begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{bmatrix} \rightarrow \begin{bmatrix} E_{11} & E_{12} & E_{13} \\ E_{21} & E_{22} & E_{23} \\ E_{31} & E_{32} & E_{33} \end{bmatrix} \begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{bmatrix} \rightarrow \begin{bmatrix} E_{11} & \nu_{12} E_{11} & \nu_{13} E_{11} \\ \nu_{21} E_{11} & E_{22} & \nu_{23} E_{11} \\ \nu_{31} E_{11} & \nu_{32} E_{11} & E_{33} \end{bmatrix}$$

Stress-Strain Relationship

$$E_{xx} = \frac{1}{\nu} E (\sigma_x - \nu (\sigma_y + \sigma_z))$$

$$E_{yy} = \frac{1}{\nu} E (\sigma_y - \nu (\sigma_x + \sigma_z))$$

$$E_{zz} = \frac{1}{\nu} E (\sigma_z - \nu (\sigma_x + \sigma_y))$$

$$\tau_{xy} = E \nu (\sigma_x - \sigma_y) / (1 + \nu)$$

Plane Stress Condition

$$E_{xx} = \frac{1}{\nu} E (\sigma_x - \nu \sigma_y)$$

$$E_{yy} = \frac{1}{\nu} E (\sigma_y - \nu \sigma_x)$$

$$E_{zz} = -\nu E (\sigma_x + \sigma_y) / (1 + \nu)$$

$$\tau_{xy} = E \nu (\sigma_x - \sigma_y) / (1 + \nu)$$

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